

1. Answer the following questions

- a) Estimate the *percent overshoot*, *settling time*, *peak time* and *rise time* of the step response given by

$$c(t) = 0.009804 - 0.0001857e^{-5.1t} - 0.009990e^{-2t} \cos(9.796t) - 0.001942e^{-2t} \sin(9.796t)$$

- b) The state-space representation of a system is given below. Find the poles of the system

$$\dot{\mathbf{x}} = \begin{bmatrix} -2 & -1 \\ -3 & -5 \end{bmatrix} \mathbf{x} + \begin{bmatrix} 1 \\ 2 \end{bmatrix} u(t)$$

$$y = [3 \quad 2] \mathbf{x}$$

- c) The transfer function of a system is given by  $G(s) = \frac{s^2 + 3s + 2}{s^4 + 5s^3 + 3s^2 + 2s + 2}$ . Give a state-space representation of the system.

- d) Consider the system shown in Fig. 1. Develop a state-space model of the system.  $m_1 = 9$  kg,  $m_2 = 1$  kg,  $k_1 = 24$  kN/m,  $k_2 = 3$  kN/m.

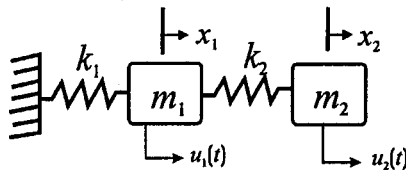


Figure 1

[4+2+3+4=13]

2. The open-loop transfer function of a system is  $G(s) = \frac{1}{s(s+2)}$ .

- a) Represent the system in controllable canonical form  
 b) Design a state-feedback controller to place the closed-loop poles at -2, -3.  
 c) Design a PID controller for placing the dominant closed-loop poles at  $-2 \pm 2j$ .

[3+8+8=19]

3. The control system of a boring machine is shown in Fig. 2 where  $R(s)$  and  $Y(s)$  are the required and achieved boring angles, respectively.  $D(s)$  is the disturbance

to the system. Determine the range of values of K for satisfying the following requirements.

- the closed-loop system is stable
- output due to the step disturbance in less than 10%
- overshoot due to step input is less than 10%

[19]

4. The open-loop transfer function of a unity feedback system is given by

$$G = \frac{K(s+2)}{(s+3)(s+1)(s^2+2s+5)}$$

- Sketch the root-locus of the system
- Determine the value of K beyond which the system become oscillatory. Also find the oscillation frequency
- Draw the Nyquist plot for K=10
- Determine the gain margin and phase margin of the system for K=10

[5+4+6+4=19]

5. Design a Positive Position Feedback Controller for assigning 40% damping ratio to both the modes of vibration of the system described in question No. 1(d) and Fig. 1.

[19]

6. Fig. 2 depicts the acceleration feedback control system of a single degree-of-freedom vibrating structure. Show that when the filter frequency is tuned to the structural natural frequency

- the condition of equal damping of the closed-loop poles is  $\zeta_f \leq \frac{K_c}{\omega_n^2}$
- the condition of equal natural frequency of the closed-loop poles is  $\zeta_f \geq \frac{K_c}{\omega_n^2}$
- Find the natural frequencies of the system when condition (a) is satisfied
- Find the closed-loop damping ratios when condition (b) is satisfied

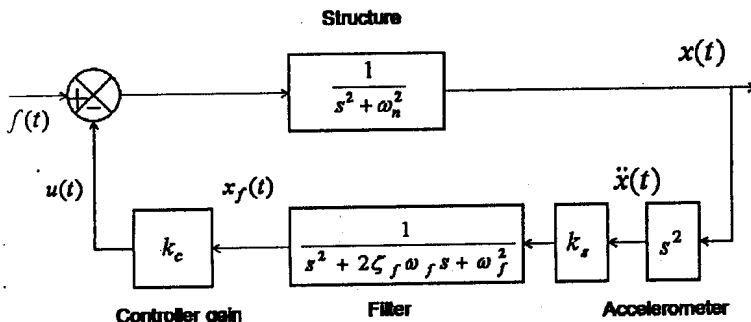


Figure 2  $K_c = k_s k_c$

[3+3+7+6=19]